## Statics Equation Sheet

$$
a x2 + b x + c = 0,
$$
  
\n
$$
x = \frac{-b \pm \sqrt{b^{2} - 4ac}}{2a}
$$
  
\n
$$
2\pi \text{ rad} \leftrightarrow 360^{\circ}
$$
  
\n
$$
\frac{\sin \theta_{a}}{a} = \frac{\sin \theta_{b}}{b} = \frac{\sin \theta_{c}}{c}
$$
  
\n
$$
c^{2} = a^{2} + b^{2} - 2ab \cos \theta_{c}
$$

 $\theta_a + \theta_b + \theta_c = \pi (180°)$  $\sin \theta = \frac{b}{c} = \frac{\text{Opposite}}{\text{Hypotenuse}}$  $\cos \theta = \frac{a}{c} = \frac{\text{Adjacent}}{\text{Hypotenuse}}$  $\tan \theta = \frac{b}{a} = \frac{\text{Opposite}}{\text{Adjacent}}$ 

 $\sin(\alpha \pm \beta) = \sin \alpha \cos \beta \pm \cos \alpha \sin \beta$  $\cos(\alpha \pm \beta) = \cos \alpha \cos \beta \mp \sin \alpha \sin \beta$  $W = mq$  $q = 32.2$  ft/sec<sup>2</sup> = 9.81 m/sec<sup>2</sup>  $N = \frac{kg \cdot m}{sec^2}$ ,  $slug = \frac{lb}{(ft/sec^2)}$ 



Chapter 2: Force Vectors

$$
\vec{R} = \sum \vec{F}, \quad \vec{F} = F\hat{\lambda}
$$
\n
$$
\hat{\lambda}_{AB} = \frac{\vec{r}_{A \to B}}{|\vec{r}_{A \to B}|}
$$
\n
$$
= \frac{\Delta x \hat{i} + \Delta y \hat{j} + \Delta z \hat{k}}{\sqrt{(\Delta x)^2 + (\Delta y)^2 + (\Delta z)^2}}
$$
\n
$$
\hat{\lambda}_{AB} = \cos \theta_x \hat{i} + \cos \theta_y \hat{j} + \cos \theta_z \hat{k}
$$
\n
$$
\hat{\lambda}_{AB} = \sin \theta_y \cos \phi \hat{i} + \cos \theta_y \hat{j}
$$
\n
$$
+ \sin \theta_y \sin \phi \hat{k}
$$
\n
$$
\cos^2 \theta_x + \cos^2 \theta_y + \cos^2 \theta_z = 1
$$
\n
$$
\vec{F} = F_x \hat{i} + F_y \hat{j}
$$
\n
$$
= F \cos \theta \hat{i} + F \sin \theta \hat{j}
$$

$$
= F \, \measuredangle \theta
$$

### Chapter 3: Equilibrium of Particles

$$
\sum \vec{F} = \left(\sum F_x\right) \hat{i} \n+ \left(\sum F_y\right) \hat{j} \n+ \left(\sum F_z\right) \hat{k} = \vec{0} \n\hat{i} : \sum F_x = 0 \n\hat{j} : \sum F_y = 0 \n\hat{k} : \sum F_z = 0
$$

### Chapter 4: Moments of a Force

The moment of *F* about point *A*

$$
\vec{M}_A = \vec{r} \times \vec{F}, \quad |M_A| = d_\perp F
$$

where  $\vec{r}$  is a position vector from  $P$  to any point on the line of action of  $\vec{F}$ .

 $M_A = (Fr) \sin \varphi = F(r \sin \varphi) = F d_\perp$  $M_A = (Fr) \sin \varphi = (F \sin \varphi) r = F_{\perp} r$ ,  $\vec{M}_A = \vec{r} \times \left( \vec{F}_1 + \vec{F}_2 \right)$  $= \vec{r} \times \vec{F_1} + \vec{r} \times \vec{F_2}$ 

Moment about a line:

$$
M_{AB} = \hat{\lambda}_{AB} \cdot \vec{M}_P
$$
  
=  $\hat{\lambda}_{AB} \cdot (\vec{r} \times \vec{F})$   
=  $|M_P| \cos \theta$ 

$$
M_{AB} = \begin{vmatrix} \lambda_x & \lambda_y & \lambda_z \\ r_x & r_y & r_z \\ F_x & F_y & F_z \end{vmatrix}
$$

Equivalent Force-Couple System:

$$
\vec{R} = \sum \vec{F},
$$
  
\n
$$
\vec{M}_O = \sum \vec{M}_O
$$
  
\n
$$
= \sum (\vec{r} \times \vec{F}) + \sum \vec{M}_{Couples}
$$

Equivalent Systems:

$$
\left(\sum \vec{F}\right)_{\substack{\text{System} \\ 1}} = \left(\sum \vec{F}\right)_{\substack{\text{System} \\ 2}} \left(\sum \vec{M}_P\right)_{\substack{\text{System} \\ 2}} \left(\sum \vec{M}_P\right)_{\substack{\text{System} \\ 2}} \left(\sum \vec{M}_P\right)_{\substack{\text{System} \\ 2}}
$$

Chapter 5: Equilibrium of Rigid Bodies

$$
\sum \vec{F} = 0, \text{ and } \sum \vec{M}_P = 0
$$
  

$$
\hat{i} : \sum F_x = 0
$$
  

$$
\hat{j} : \sum F_y = 0
$$
  

$$
\hat{k} : \sum F_z = 0
$$
  

$$
\hat{i} : \sum M_x = 0
$$
  

$$
\hat{j} : \sum M_y = 0
$$
  

$$
\hat{k} : \sum M_z = 0
$$

2D Planar:

$$
\rightarrow \sum F_x = 0
$$

$$
\uparrow \sum F_y = 0
$$

$$
\uparrow \sum M_P = 0
$$

Reaction forces prevent translation; Reaction moments prevent rotation

### Chapter 7: Internal Forces and Moments

In two-dimensions:

- $N =$  Normal Axial Force  $V =$ Transverse Shear Force
- $M =$ Bending Moment

$$
\frac{dV}{dx} = -w, \quad \frac{dM}{dx} = V
$$

$$
\frac{d^2M}{dx^2} = -w(x)
$$

Concentrated force *P*, shear discontinuity (jump):

$$
V^+ - V^- = -P
$$

Concentrated couple *M<sup>C</sup>* , moment jump:

$$
M^+ - M^- = -M_C
$$

In three-dimensions:

 $N_x =$  Axial Force  $V_y$  = Shear Force in *y*-direction  $V_z$  = Shear Force in *z*-direction  $T_x$  = Torque about the *x*-axis  $M_y$  = Moment about the *y*-axis  $M_z$  = Moment about the *z*-axis

#### Distributed loads *w*(*x*):

$$
F = \int_{L} w(x) dx
$$

$$
F \bar{x} = M = \int_{L} x w(x) dx
$$

Effect of  $w(x)$  is statically equivalent to resultant *F* at position  $\bar{x}$ .

### Chapter 8: Friction

$$
F_f \le (F_f)_{\text{max static}} = \mu_s F_N
$$

If at the max limit, on the verge of sliding and impending motion,

$$
F_f = \mu_s F_N
$$

$$
Belt Friction: \quad \frac{T_2}{T_1} = e^{\mu \beta}
$$

### Spring Forces and Torques

$$
F_s = k_L (L' - L)
$$

# $M_s = k_\theta (\theta' - \theta)$

## Chapter 11: Virtual Work

$$
\delta U = \delta \vec{u} \cdot \vec{F} + \delta \vec{\theta} \cdot \vec{M} = 0
$$

Virtual Power:

$$
\delta V = \delta \vec{v} \cdot \vec{F} + \delta \vec{\omega} \cdot \vec{M} = 0
$$

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## Cross (Vector) Products

- 1.  $\vec{u} \times \vec{v} = -\vec{v} \times \vec{u}$
- 2.  $a(\vec{u} \times \vec{v}) = (a\vec{u}) \times \vec{v} = \vec{u} \times (a\vec{v})$ , and <sup>w</sup>
- 3.  $\vec{u} \times (\vec{v} + \vec{w}) = \vec{u} \times \vec{v} + \vec{u} \times \vec{w}$



−  $\hat{i} \times \hat{i} = \vec{0}$   $\hat{i} \times \hat{j} = \hat{k}$  $\hat{k}$   $\hat{i} \times \hat{k} = -\hat{j}$  $\hat{j} \times \hat{i} = -\hat{k}$   $\hat{j} \times \hat{j} = \vec{0}$   $\hat{j} \times \hat{k} = \hat{i}$  $\hat{k} \times \hat{i} = \hat{j}$   $\hat{k}$  $\hat{k} \times \hat{j} = -\hat{i} \quad \hat{k} \times \hat{k} = \vec{0}$ 

$$
\vec{M} = \vec{r} \times \vec{F} = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ r_x & r_y & r_z \\ F_z & F_y & F_z \end{vmatrix}
$$

$$
= \begin{vmatrix} r_y & r_z \\ F_y & F_z \end{vmatrix} \hat{i}
$$

$$
- \begin{vmatrix} r_x & r_z \\ F_x & F_z \end{vmatrix} \hat{j}
$$

$$
+ \begin{vmatrix} r_x & r_y \\ F_x & F_y \end{vmatrix} \hat{k}
$$

$$
= (r_y F_z - r_z F_y) \hat{i}
$$

$$
- (r_x F_z - r_z F_x) \hat{j}
$$

$$
+ (r_x F_y - r_y F_x) \hat{k}
$$

In 2D plane:

$$
\vec{M} = (r_x \hat{i} + r_y \hat{j}) \times (F_x \hat{i} + F_y \hat{j})
$$
\n
$$
\vec{M} = (F_y r_x - F_x r_y) \hat{k}
$$
\n
$$
M = (F_y f_x - F_x r_y) \hat{k}
$$
\n
$$
\begin{bmatrix} M_x \\ M_y \\ M_z \end{bmatrix} = \begin{bmatrix} 0 & -z & y \\ z & 0 & -x \\ -y & x & 0 \end{bmatrix} \begin{bmatrix} F_x \\ F_y \\ F_z \end{bmatrix}
$$
\n
$$
= \begin{bmatrix} yF_z - zF_y \\ zF_x - xF_z \\ xF_y - yF_x \end{bmatrix}
$$

## Coordinate Rotations

 $x' = x \cos \theta + y \sin \theta$  $y' = -x \sin \theta + y \cos \theta$ .  $\vec{u} = u_x \hat{i} + u_y \hat{j} = u_{x'} \hat{i}' + u_{y'} \hat{j}'$  $u_{x'} = u_x \cos \theta + u_y \sin \theta$  $u_{y'} = -u_x \sin \theta + u_y \cos \theta$ 

## Dot (Scalar) Products

$$
\vec{u} \cdot \vec{v} = |\vec{u}||\vec{v}| \cos \theta
$$
  
where  $\theta$  is the angle between  $\vec{u}$  and  $\vec{v}$ .  
1.  $\vec{u} \cdot \vec{v} = \vec{v} \cdot \vec{u}$   
2.  $a(\vec{u} \cdot \vec{v}) = (a\vec{u}) \cdot \vec{v} = \vec{u} \cdot (a\vec{v})$ , and  
3.  $\vec{u} \cdot (\vec{v} + \vec{w}) = \vec{u} \cdot \vec{v} + \vec{u} \cdot \vec{w}$ 

Since  $\hat{i}, \hat{j}$ , and  $\hat{k}$  are mutually orthogonal:

 $\hat{i} \cdot \hat{i} = 1$   $\hat{i} \cdot \hat{j} = 0$   $\hat{i} \cdot \hat{k} = 0$  $\hat{j}\cdot\hat{i}=0$   $\hat{j}\cdot\hat{j}=1$   $\hat{j}\cdot\hat{k}=0$  $\hat{k} \cdot \hat{i} = 0 \quad \hat{k} \cdot \hat{j} = 0 \quad \hat{k} \cdot \hat{k} = 1$  $\vec{u} \cdot \vec{v} = \left( u_x \hat{\imath} + u_y \hat{\jmath} + u_z \hat{k} \right)$ *·*  $\left(v_x\hat{i} + v_y\hat{j} + v_z\hat{k}\right)$  $= u_xv_x + u_yv_y + u_zv_z$ 

Chapter 9: Centroids of Areas

$$
A = \sum_{i=1}^{n} A_i = \int_A dA
$$
  

$$
A\overline{x} = \sum_{i=1}^{n} \overline{x}_i A_i = \int_A x dA
$$
  

$$
A\overline{y} = \sum_{i=1}^{n} \overline{y}_i A_i = \int_A y dA
$$

Chapter 10: Area Moment of Inertia

$$
I_x = k_x^2 A = \int_A y^2 dA
$$
  

$$
I_y = k_y^2 A = \int_A x^2 dA
$$
  

$$
J_O = k_0^2 A = \int_A r^2 dA = I_x + I_y
$$

Radius of gyration:

$$
k_x, k_y, k_0 = \sqrt{k_x^2 + k_y^2}
$$
  

$$
I_{\text{rect}} = \frac{1}{12}bh^3, \quad I_{\text{circle}} = \frac{1}{4}\pi r^4
$$

 $I = \bar{I} + A d_{\perp}^2$ 

*d*<sup>⊥</sup> = perpendicular distance between parallel axes.

### Chapter 9: Center of Mass and Gravity

$$
m = \sum_{i=1}^{n} m_i = \int_m dm
$$
  
\n
$$
m\,\bar{x} = \sum_{i=1}^{n} \bar{x}_i m_i = \int_m x \,dm
$$
  
\n
$$
m\,\bar{y} = \sum_{i=1}^{n} \bar{y}_i m_i = \int_m y \,dm
$$
  
\n
$$
m\,\bar{z} = \sum_{i=1}^{n} \bar{z}_i m_i = \int_m z \,dm
$$

For center of gravity, change "*m*" to weight "*W*". For the centroid of a volume or line, change " $m$ " to " $V$ " or " $L$ ", respectively.

### Chapter 10: Mass Moment of Inertia

$$
I = k^2 m = \int_m r^2 \, \mathrm{d}m
$$

Parallel Axis Theorem:

$$
I=\bar{I}+md_\perp^2
$$

*d*<sup>⊥</sup> = perpendicular distance between parallel axes.

$$
I_{\text{bar}} = \frac{1}{12}m\ell^2, \quad m = \rho A\ell
$$

Thin plate with surface area *A*:

$$
I_{x-\text{axis}} = \rho t_z I_x
$$
  
\n
$$
I_{y-\text{axis}} = \rho t_z I_y
$$
  
\n
$$
I_{z-\text{axis}} = \rho t_z J_0 = \rho t_z (I_x + I_y)
$$

where  $I_x, I_y, J_0$  are area moments of inertia.



